

PATENT ABSTRACTS OF JAPAN

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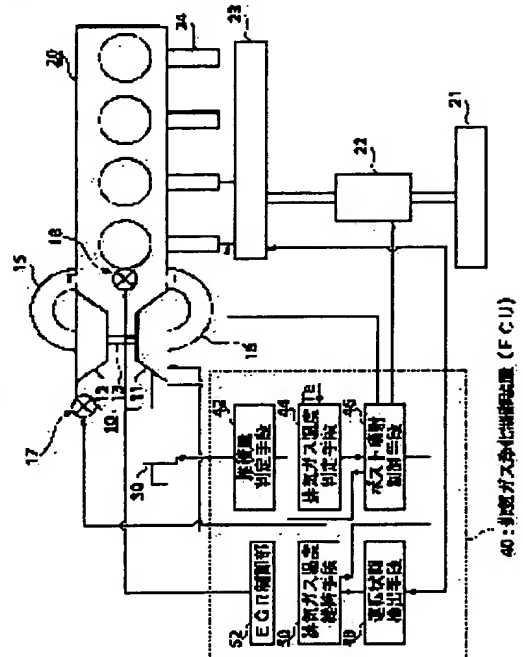
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(54) EXHAUST EMISSION CONTROL DEVICE AND EXHAUST EMISSION CONTROL METHOD

(57)Abstract:

PROBLEM TO BE SOLVED: To effectively utilize thermal energy generated in a post injection.

SOLUTION: This exhaust emission control device comprises: a post injection control means 46 which starts the post injection of a diesel engine 20 when a collection amount determination means 42 determinates that a particulate collection amount of a particulate filter 30 exceeds a predetermined threshold value and an exhaust gas temperature (T_e) is lower than a predetermined post injection start temperature (T_{er}); an operating condition detection means 48 which detects an operating condition of the diesel engine 20; and an exhaust gas temperature maintaining means 50 which prevents a rotation speed of a turbocharger 10 from increasing when an engine speed of the diesel engine 20 detected in the operating condition detection means 48 is not more than middle speed, and the post injection is carried out in a low load operation condition.



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CLAIMS

[Claim(s)]

[Claim 1] It has the particulate filter which carries out uptake of the particulate contained in the exhaust gas of the diesel power plant which has a turbocharger, and said diesel power plant. An alimentation judging means to judge whether the particulate alimentation of the; aforementioned particulate filter exceeded the predetermined threshold in the exhaust gas purge used for the engine which reproduces said particulate filter with the heat energy of said exhaust gas, An exhaust gas temperature judging means to judge whether the exhaust gas temperature (Te) of said diesel power plant is lower than predetermined postinjection initiation temperature (Ter); Said particulate alimentation exceeds a predetermined threshold. And when said exhaust gas temperature is lower than said postinjection initiation temperature Postinjection of said diesel power plant The postinjection control means to start and an operational status detection means to detect the operational status of the; aforementioned diesel power plant; when the rotational frequency of said diesel power plant detected with said operational status detection means performs postinjection by below medium speed and the operational status of low loading ; exhaust gas purge possessing the exhaust gas temperature maintenance means operated so that rotational frequency increase of said turbocharger may be prevented.

[Claim 2] Said exhaust gas temperature maintenance means is an exhaust gas purge given in; claim 1 which opens fully the adjustable nozzle or way SUTOGETO BARUBU prepared in said turbocharger when the engine speed of said diesel power plant performs postinjection by below medium speed and the operational status of low loading.

[Claim 3] An exhaust gas purge given in; claim 1 or claim 2 whose operational status of below medium speed and low loading the engine speed of said diesel power plant which operates said exhaust gas temperature maintenance means is the idling of said diesel power plant.

[Claim 4] An exhaust gas purge given in any of; claim 1 which consists of or or larger making a judgment than predetermined threshold differential pressure (dPr) thru/or claim 3 said alimentation judging means is in the differential pressure (dP) of the entrance-side exhaust pressure of a particulate filter, and an outlet side exhaust pressure.

[Claim 5] It has the particulate filter which carries out uptake of the particulate contained in the exhaust gas of the diesel power plant which has a turbocharger, and said diesel power plant. Said particulate filter with the heat energy of said exhaust gas In the exhaust gas purification approach used for the engine to reproduce ; Said particulate alimentation is larger than a predetermined threshold. or [that the particulate alimentation of said particulate filter exceeded the predetermined threshold] -- judging --; -- or [that the exhaust gas temperature (Te) of said diesel power plant is lower than predetermined postinjection initiation temperature (Ter)] -- judging --; -- And when said exhaust gas temperature is lower than said postinjection initiation temperature The; exhaust gas purification approach which postinjection of said diesel power plant is started and the engine speed of the; aforementioned diesel power plant judges in the operational status of below medium speed and low loading, and prevents engine-speed increase of said turbocharger when the engine speeds of the; aforementioned diesel power plant are below medium speed and low loading.

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DETAILED DESCRIPTION

[Detailed Description of the Invention]

[0001]

[Field of the Invention] This invention relates to the exhaust gas purge and the exhaust gas purification approach of reproducing DPF (diesel particulate filter) efficiently especially also at the time of low loading like an idling about the equipment which carries out uptake of the minute solid particulate (particulate) contained in the exhaust gas of a diesel power plant, and incinerates it.

[0002]

[Description of the Prior Art] Compression ignition of the injected fuel is carried out by the diesel power plant. Moreover, since a particle will surely occur in exhaust gas if heterogeneous gaseous mixture is burned, it is next to impossible to design the diesel power plant out of which soot does not come theoretically. However, by the diesel power plant, particulate discharge is decreased below to the visible limitation by adopting a turbocharger (especially with an intercooler), for example.

[0003] However, need increase of the shipping freight traffic in a big city has caused overcrowding of an automobile. Then, even if one set of an one-set diesel-power-plant engine performs the cure against exhaust gas, since a lot of diesel-power-plant engines are operating, particulate total emission increases and environmental load-carrying capacity is exceeded in a huge overpopulated city in many cases. Then, in order to remove a particulate out of the exhaust gas of a diesel power plant, the car which carried the diesel power plant is equipped with the particulate filter.

[0004] The particulate filter is equipped with the porous wall which has not coated honeycomb structure. The particulate contained in exhaust gas is removed by getting it blocked in a part for the pore of a porous wall. Since the particulate by which uptake was carried out accumulates on a particulate filter, it is necessary to remove this deposited particulate at fixed spacing, and to reproduce a particulate filter. For example, the heat-treating method is used as this playback system. The heat-treating method connects the heater for particulate combustion of high power, raises the temperature of exhaust gas to about 700 degrees C, and burns a particulate.

[0005] However, since it causes aggravation of fuel consumption since this heat-treating method needs the heating energy of a heater, and it consists of two lines, playback and uptake, it has the technical problem that equipment becomes complicated greatly. Then, the equipment called the continuation playback mold DPF is developed briskly recently. This equipment lowers a particulate combustion temperature according to a catalyst, and reproduces it continuously with the heat energy of exhaust gas. Since it is not necessary to make it two lines again, it is more advantageous than not compulsive heating at a heater but DPF using the heat-treating method in the point of fuel consumption and a layout.

[0006]

[Problem(s) to be Solved by the Invention] However, playback is not performed in the degree region of low temperature where a catalyst does not activate the above-mentioned continuation playback mold DPF. Therefore, a particulate continued accumulating on the filter, exhaust gas pressure went up by the blinding of a filter, without being reproduced at the time of the low idling condition of an exhaust-gas temperature, or downward slope transit, and the technical problem that trouble was caused to the power engine performance occurred. Then, raising an exhaust-gas temperature to the activity temperature region of a catalyst by postinjection in such a case, and reproducing compulsorily is performed. However, with the engine equipped with the turbocharger, even if it raises an exhaust-gas temperature by postinjection with much trouble, it will be used for the work from which this heat energy raises the engine speed of a turbine, and has the technical problem that it will fall, the outlet temperature, i.e., the DPF inlet temperature, of a turbine. This invention solves the technical problem mentioned above, and it aims at providing an effective target with an available exhaust gas purge for the heat energy generated in postinjection.

[0007]

[Means for Solving the Problem] In order to attain the above-mentioned purpose, as shown in drawing 1, the exhaust gas purge by this invention is equipped with the particulate filter 30 which carries out uptake of the particulate contained in the exhaust gas of the diesel power plant 20 which has a turbocharger 10, and a diesel power plant 20, and is used for the engine which reproduces a particulate filter 30 with the heat energy of said exhaust gas.

[0008] An alimention judging means 42 to judge whether said exhaust gas purge exceeded the threshold predetermined in the particulate alimention of a particulate filter 30, An exhaust gas temperature judging means 44 to judge whether the exhaust gas temperature (Te) of a diesel power plant 20 is lower than predetermined postinjection initiation temperature (Ter), The postinjection control means 46 which said particulate alimention exceeds a predetermined threshold, and starts postinjection of a diesel power plant 20 when said exhaust gas temperature is lower than said postinjection initiation temperature, An operational status detection means 48 to detect the operational status of a diesel power plant 20, When the engine speed of the diesel power plant 20 detected with the operational status detection means 48 performs postinjection by below medium speed and the operational status of low loading, the exhaust gas temperature maintenance means 50 operated so that engine-speed increase of a turbocharger 10 may be prevented is provided.

[0009] Thus, it is good to consider as the configuration which it sets to the constituted exhaust gas purge, and the alimention judging means 42 judges whether the particulate alimention of a particulate filter 30 exceeded the predetermined threshold, and judges whether the differential pressure (dP) of the entrance-side exhaust pressure of a particulate filter 30 and an outlet side exhaust pressure is larger than predetermined threshold differential pressure (dPr). It judges whether the exhaust gas temperature judging means 44 has the exhaust gas temperature (Te) of a diesel power plant 20 lower than predetermined postinjection initiation temperature (Ter). The particulate alimention of a particulate filter 30 exceeds a predetermined threshold, and the postinjection control means 46 starts postinjection of a diesel power plant 20, when the exhaust gas temperature of a diesel power plant 20 is lower than postinjection initiation temperature, and it makes exhaust gas temperature high to the regenerating temperature of a particulate filter 30.

[0010] The operational status detection means 48 detects the operational status of a diesel power plant 20, and the engine speeds of a

diesel power plant 20 are below medium speed and the operational status of low loading like an idling, for example, it has detected it in the condition that the need of the high load is carried out to the diesel power plant 20 like [at the time of acceleration]. The rotational frequencies of a diesel power plant 20 are below medium speed and the operational status of low loading, and when postinjection is performed by the postinjection control means 46, the exhaust gas temperature maintenance means 50 prevents rotational frequency increase of a turbocharger 10, burns the particulate deposited on the particulate filter 30, and is reproduced. As a configuration which prevents engine-speed increase of a turbocharger 10, you open fully the adjustable nozzle or way SUTOGETOBARUBU prepared, for example in the turbocharger 10, and it is made for the energy of exhaust gas not to make it consume to the turbine side of a turbocharger 10. On the other hand, when judged as the operational status by which the above output is needed to some extent for a diesel power plant 20 with the operational status detection means 48, it increases and the rotational frequency of a turbocharger 10 is made into high power.

[0011] In order to attain the above-mentioned purpose, the exhaust gas purification approach by this invention judges whether the particulate alimention of a particulate filter 30 exceeded the predetermined threshold first, as shown in drawing 7 (S102). For example, by judging whether the differential pressure (dP) of the entrance-side exhaust pressure of a particulate filter 30 and an outlet side exhaust pressure is larger than predetermined threshold differential pressure (dPr), particulate alimention is measured indirectly. Next, it judges whether the exhaust gas temperature (Te) of a diesel power plant 20 is lower than predetermined postinjection initiation temperature (Ter) (S104). And more greatly than a threshold predetermined in particulate alimention, when exhaust gas temperature is lower than postinjection initiation temperature, postinjection of a diesel power plant 20 is started (S106). Next, it judges whether the rotational frequencies of a diesel power plant 20 are below medium speed and the operational status of low loading (S108), and in corresponding, a measure required to prevent rotational frequency increase of a turbocharger is taken, and it opens fully the adjustable nozzle or way SUTOGETOBARUBU prepared in the turbocharger 10 (S110).

[0012]

[Embodiment of the Invention] Hereafter, the gestalt of operation of this invention is explained with reference to a drawing. In addition, in each drawing, the same or the explanation which attached the same sign or the similar sign at the corresponding member, and overlapped is omitted mutually. Drawing 1 is a block diagram explaining the gestalt of operation of the exhaust gas purge used for the diesel power plant of this invention.

[0013] In drawing, the fuel stored in the fuel tank 21 is stored in the fuel rail 23 with a pump 22. The fuel rail 23 is also called a common rail, it is a form independent of an engine speed and the fuel amount of supply, and generates an injection pressure and has heightened the average injection pressure. An injector 24 injects the opening time defined by ECU (Electric Control Unit) and the fuel oil consumption which becomes settled from the fuel pressure from the fuel rail 23 to the combustion chamber of a diesel power plant 20.

[0014] The turbocharger 10 is equipped with the exhaust air drive turbine 11 which transforms the energy in the exhaust gas of a diesel power plant 20 into mechanical energy, and the compressor 12 which compresses inhalation air. While specific fuel consumption is remarkably improved with the engine using a turbocharger 10 as compared with the natural aspiration engine of the same output, exhaust gas concentration is also improved. A shaft 13 conducts rotation of the exhaust air drive turbine 11 to a compressor 12.

[0015] An inlet manifold 15 is a duct for supplying the air compressed by the compressor 12 to the combustion chamber of a diesel power plant 20. An exhaust manifold 16 is a duct for supplying exhaust gas to the exhaust air drive turbine 11 from the combustion chamber of a diesel power plant 20. The intake throttle 17 is the throttle valve of the inhalation air of a compressor 12. EGR valve 18 is a bulb which is prepared in the last section of an EGR (Exhaust Gas Recirculation) path, and controls the amount of exhaust gas recycling. Here, EGR introduces and carries out recycling of a part of exhaust gas to an inhalation network, by mixing in gaseous mixture, lowers combustion temperature and reduces the yield of Nox. DPF30 is a particulate filter, needs to remove the deposited particulate at fixed spacing, and needs to reproduce a particulate filter while it carries out uptake of the particulate contained in exhaust gas.

[0016] ECU is equipped with the program required for the electronic circuitry as an exhaust gas purification control unit 40, or actuation of a microprocessor while it performs the combustion control of a diesel power plant 20. The exhaust gas purification control unit 40 equips the alimention judging means 42, the exhaust gas temperature judging means 44, the postinjection control means 46, the operational status detection means 48, the exhaust gas temperature maintenance means 50, and the list with the EGR control section 52.

[0017] The alimention judging means 42 judges whether the particulate alimention of a particulate filter 30 exceeded the predetermined threshold, and judges whether the differential pressure dP of the entrance-side exhaust pressure of a particulate filter 30 and an outlet side exhaust pressure is larger than the predetermined threshold differential pressure dPr. The predetermined threshold differential pressure dPr judges that the particulate amount deposited on the particulate filter 30 reached to the reproductive complement. the differential pressure dP of the entrance-side exhaust pressure of a particulate filter 30, and an outlet side exhaust pressure -- differential pressure sensors, such as a semi-conductor type, -- dependability -- it can measure highly. In addition, the particulate alimention of a particulate filter 30 may be replaced with a differential pressure type, and you may measure by the weight formula, and a deposition condition may be checked by looking using a visual sensor and it may measure that the particulate amount of the exhaust gas which penetrated the particulate filter 30 further is not decreasing enough.

[0018] It judges whether the exhaust gas temperature judging means 44 has exhaust gas temperature Te of a diesel power plant 20 lower than the predetermined postinjection initiation temperature Ter. Measurement of exhaust gas temperature Te is measured by the sensor from which resistance changes according to temperature like a resistance bulb, and is measured with the thermometer of other formats like an infrared type thermometer. The postinjection initiation temperature Ter is defined on the basis of exhaust gas temperature required for playback of a particulate filter 30.

[0019] The particulate alimention of a particulate filter 30 exceeds a predetermined threshold, and the postinjection control means 46 starts postinjection of a diesel power plant 20, when the exhaust gas temperature of a diesel power plant 20 is lower than postinjection initiation temperature. Drawing 2 is the timing chart of the fuel injection injected by the combustion chamber of a diesel power plant from an injector. There are pilot injection, Main injection, and postinjection as timing of fuel injection. The timing of fuel injection is defined by ECU.

[0020] The operational status detection means 48 detects the operational status of a diesel power plant 20, and the engine speeds of a diesel power plant 20 are below medium speed and the operational status of low loading like [at the time of an idling or downward slope transit], for example, it has detected it in the condition that the need of the high load is carried out to the diesel power plant 20 like [at the time of acceleration]. Drawing 3 is the explanatory view of the operational status of a diesel power plant, Torque Nm is shown on an axis of ordinate, and engine-speed rpm is shown on the axis of abscissa. a operating range E1 -- the high engine-speed field of a diesel power plant, and a list -- an inside low engine-speed field -- it is -- the crown -- the usual operating range of the diesel power plant at the time of running a truck and a bus is shown including the torque field. The rotational frequencies of a diesel power

plant 20 are below medium speed and the operational status of low loading like an idling or moderation operation at the time of a light load in the medium-speed low torque field E2.

[0021] It returns to drawing 1 again and explanation is continued. The exhaust gas temperature raised by postinjection prevents it being changed into the mechanical energy of a turbocharger 10, and falling again, burns the particulate deposited on the particulate filter 30, and reproduces the exhaust gas temperature maintenance means 50. Since exhaust gas temperature is lower than the temperature suitable for playback of a particulate filter when an engine load like an idling is light, it is necessary to raise exhaust gas temperature to the temperature suitable for playback of a particulate filter. Then, it checks whether the rotational frequencies of a diesel power plant 20 are below medium speed and the operational status of low loading with the operational status detection means 48. When judged as the operational status by which the above output is needed to some extent for a diesel power plant 20 with the operational status detection means 48, it increases and the rotational frequency of a turbocharger 10 is made into high power.

[0022] When it is checked that the rotational frequencies of a diesel power plant 20 are below medium speed and the operational status of low loading and postinjection is further performed by the operational status detection means 48, the exhaust gas temperature maintenance means 50 prevents rotational frequency increase of a turbocharger 10, and prevents rotational frequency increase of a turbocharger 10. You specifically open fully the adjustable nozzle or way SUTOGETOBARUBU prepared in the turbocharger 10, and it is made for the energy of exhaust gas not to make it consume to the turbine side of a turbocharger 10.

[0023] Drawing 4 is the block diagram of an adjustable blade type turbine. As for the adjustable blade type turbine, the guide blade 64 as an adjustable nozzle which can be adjusted is formed near the air inhalation opening 66 of the turbine housing 61. The guide blade 64 which can be adjusted is controlled according to the exhaust gas flow rate and gas rate which flow to a turbine impeller, and a turbine rotational frequency is maintained in a supercharge scope. Rotation of an adjust ring 62 can adjust easily the include angle of the guide blade 64 which can be adjusted. Adjustment of an include angle is performed by the cone roll cam 63 or the adjusting lever 65 attached to each blade.

[0024] Drawing 5 is the explanatory view of the charge pressure adjustment by way SUTOGETOBARUBU. Way SUTOGETOBARUBU 73 was formed in the exhaust pipe 74 which connects an engine 71 and a turbocharger 72, and has changed the condition of performing an air bypass and discharging the exhaust gas of an engine 71 in the direct open air without a turbocharger 72, and the condition of supplying a turbocharger 72.

[0025] Drawing 6 is drawing which explains relation with the working state of an exhaust gas temperature maintenance means to the existence list of exhaust gas temperature and postinjection. ** The bar graph shown by "W/o post injection" shows the condition of not performing postinjection, and exhaust gas temperature is the lowest. ** Although the condition of performing postinjection is shown and exhaust gas temperature is higher than the condition of not performing postinjection, the bar graph shown by "W/post injection" does not have the enough rise of exhaust gas temperature, in order that a turbocharger may work. ** The bar graph shown by "W/post injection & full open VNT or WG" shows the condition of opening fully the adjustable nozzle or way SUTOGETOBARUBU (WG) prepared in the turbocharger 10 by the exhaust gas temperature maintenance means while performing postinjection, and exhaust gas temperature is the highest. It becomes the temperature which exhaust gas temperature burns the particulate deposited on the particulate filter 30, and can reproduce by using together postinjection and an exhaust gas temperature maintenance means.

[0026] Thus, actuation of the constituted equipment is explained below. Drawing 7 is a flow chart explaining actuation of the equipment of drawing 1. First, the exhaust gas purification control device 40 starts the regeneration program of a particulate filter prepared in ECU a fixed period (S100). It judges whether the differential pressure (dP) of the entrance-side exhaust pressure of a particulate filter 30 and an outlet side exhaust pressure is larger than predetermined threshold differential pressure (dPr) (S102). Here, the particulate alimentation of a particulate filter 30 judges indirectly whether the predetermined threshold was exceeded by measuring the differential pressure of the entrance-side exhaust pressure of a particulate filter 30, and an outlet side exhaust pressure.

[0027] Next, it judges whether the exhaust gas temperature (Te) of a diesel power plant 20 is lower than predetermined postinjection initiation temperature (Ter) (S104). And more greatly than a threshold predetermined in particulate alimentation, when exhaust gas temperature is lower than postinjection initiation temperature, postinjection of a diesel power plant 20 is started (S106). Next, it judges whether the rotational frequencies of a diesel power plant 20 are below medium speed and the operational status of low loading (S108). Specifically, it judges whether there are few engine speeds (Rpm) rather than the predetermined engine speed (Rpmr) as which the total fuel injection quantity (Q) by the injector 24 was similarly beforehand determined by experiment few from the predetermined fuel oil consumption (Qr) beforehand defined by experiment. And in corresponding to S108, it opens fully the adjustable nozzle or way SUTOGETOBARUBU prepared in the turbocharger 10 (S110). And it escapes from processing (S112).

[0028]

[Effect of the Invention] As explained above, since the exhaust gas purge of this invention judges whether the predetermined threshold used as those criteria for which the particulate alimentation of a particulate filter needs playback with an alimentation judging means was exceeded, the stage when a particulate filter should be reproduced can judge it exactly. Since postinjection of a diesel power plant is started by the postinjection control means while exhaust gas temperature Te of a diesel power plant judges whether it is lower than the predetermined postinjection initiation temperature Ter with an exhaust gas temperature judging means, exhaust gas temperature can be held to the suitable temperature for particulate filter playback. Since it is controlling so that the exhaust gas temperature which prevented engine-speed increase of a turbocharger and rose by postinjection is not lost with a turbocharger when the engine speed of a diesel power plant performs postinjection by below medium speed and the operational status of low loading with an exhaust gas temperature maintenance means, a particulate filter is reproducible.

[0029] For example, the time of an idling has the rotational frequency of a diesel power plant as below medium speed and operational status of low loading. For example, at the time of an idling, to being about 20 [mg/m³], a particulate yield is about 200 [mg/m³], and there is an about 10 times as many aperture as this depending on the operation mode of a diesel-power-plant car at the time of the maximum output. although the exhaust gas temperature at the time of an idling is markedly boiled as compared with 100-200 degrees C and 550-750 degrees C at the time of the maximum output and is low temperature, it can be made to go up to the temperature to which exhaust gas temperature of the operational status of below medium speed and low loading is made as for playback of a particulate filter by the engine speed of a diesel power plant by operation of this invention Then, without being used for the work from which the heat energy by postinjection raises the rotational frequency of a turbine in the time of the low idling condition of an exhaust-gas temperature, or downward slope transit, the engine equipped with the turbocharger can also raise an exhaust-gas temperature to the activity temperature region of a catalyst by postinjection, and can reproduce a particulate filter compulsorily.

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[Description of the Prior Art] Compression ignition of the injected fuel is carried out by the diesel power plant. Moreover, since a particle will surely occur in exhaust gas if heterogeneous gaseous mixture is burned, it is next to impossible to design the diesel power plant out of which soot does not come theoretically. However, by the diesel power plant, particulate discharge is decreased below to the visible limitation by adopting a turbocharger (especially with an intercooler), for example.

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[0004] The particulate filter is equipped with the porous wall which has not coated honeycomb structure. The particulate contained in exhaust gas is removed by getting it blocked in a part for the pore of a porous wall. Since the particulate by which uptake was carried out accumulates on a particulate filter, it is necessary to remove this deposited particulate at fixed spacing, and to reproduce a particulate filter. For example, the heat-treating method is used as this playback system. The heat-treating method connects the heater for particulate combustion of high power, raises the temperature of exhaust gas to about 700 degrees C, and burns a particulate.

[0005] However, since it causes aggravation of fuel consumption since this heat-treating method needs the heating energy of a heater, and it consists of two lines, playback and uptake, it has the technical problem that equipment becomes complicated greatly. Then, the equipment called the continuation playback mold DPF is developed briskly recently. This equipment lowers a particulate combustion temperature according to a catalyst, and reproduces it continuously with the heat energy of exhaust gas. Since it is not necessary to make it two lines again, it is more advantageous than not compulsive heating at a heater but DPF using the heat-treating method in the point of fuel consumption and a layout.

[0006]

[Problem(s) to be Solved by the Invention] However, playback is not performed in the degree region of low temperature where a catalyst does not activate the above-mentioned continuation playback mold DPF. Therefore, a particulate continued accumulating on the filter, exhaust gas pressure went up by the blinding of a filter, without being reproduced at the time of the low idling condition of an exhaust-gas temperature, or downward slope transit, and the technical problem that trouble was caused to the power engine performance occurred. Then, raising an exhaust-gas temperature to the activity temperature region of a catalyst by postinjection in such a case, and reproducing compulsorily is performed. However, with the engine equipped with the turbocharger, even if it raises an exhaust-gas temperature by postinjection with much trouble, it will be used for the work from which this heat energy raises the engine speed of a turbine, and has the technical problem that it will fall, the outlet temperature, i.e., the DPF inlet temperature, of a turbine. This invention solves the technical problem mentioned above, and it aims at providing an effective target with an available exhaust gas purge for the heat energy generated in postinjection.

[0007]

[Means for Solving the Problem] In order to attain the above-mentioned purpose, as shown in drawing 1, the exhaust gas purge by this invention is equipped with the particulate filter 30 which carries out uptake of the particulate contained in the exhaust gas of the diesel power plant 20 which has a turbocharger 10, and a diesel power plant 20, and is used for the engine which reproduces a particulate filter 30 with the heat energy of said exhaust gas.

[0008] An alimention judging means 42 to judge whether said exhaust gas purge exceeded the threshold predetermined in the particulate alimention of a particulate filter 30, An exhaust gas temperature judging means 44 to judge whether the exhaust gas temperature (Te) of a diesel power plant 20 is lower than predetermined postinjection initiation temperature (Ter), The postinjection control means 46 which said particulate alimention exceeds a predetermined threshold, and starts postinjection of a diesel power plant 20 when said exhaust gas temperature is lower than said postinjection initiation temperature, An operational status detection means 48 to detect the operational status of a diesel power plant 20, When the engine speed of the diesel power plant 20 detected with the operational status detection means 48 performs postinjection by below medium speed and the operational status of low loading, the exhaust gas temperature maintenance means 50 operated so that engine-speed increase of a turbocharger 10 may be prevented is provided.

[0009] Thus, it is good to consider as the configuration which it sets to the constituted exhaust gas purge, and the alimention judging means 42 judges whether the particulate alimention of a particulate filter 30 exceeded the predetermined threshold, and judges whether the differential pressure (dP) of the entrance-side exhaust pressure of a particulate filter 30 and an outlet side exhaust pressure is larger than predetermined threshold differential pressure (dPr). It judges whether the exhaust gas temperature judging means 44 has the exhaust gas temperature (Te) of a diesel power plant 20 lower than predetermined postinjection initiation temperature (Ter). The particulate alimention of a particulate filter 30 exceeds a predetermined threshold, and the postinjection control means 46 starts postinjection of a diesel power plant 20, when the exhaust gas temperature of a diesel power plant 20 is lower than postinjection initiation temperature, and it makes exhaust gas temperature high to the regenerating temperature of a particulate filter 30.

[0010] The operational status detection means 48 detects the operational status of a diesel power plant 20, and the engine speeds of a

diesel power plant 20 are below medium speed and the operational status of low loading like an idling, for example, it has detected it in the condition that the need of the high load is carried out to the diesel power plant 20 like [at the time of acceleration]. The rotational frequencies of a diesel power plant 20 are below medium speed and the operational status of low loading, and when postinjection is performed by the postinjection control means 46, the exhaust gas temperature maintenance means 50 prevents rotational frequency increase of a turbocharger 10, burns the particulate deposited on the particulate filter 30, and is reproduced. As a configuration which prevents engine-speed increase of a turbocharger 10, you open fully the adjustable nozzle or way SUTOGETOBARUBU prepared, for example in the turbocharger 10, and it is made for the energy of exhaust gas not to make it consume to the turbine side of a turbocharger 10. On the other hand, when judged as the operational status by which the above output is needed to some extent for a diesel power plant 20 with the operational status detection means 48, it increases and the rotational frequency of a turbocharger 10 is made into high power.

[0011] In order to attain the above-mentioned purpose, the exhaust gas purification approach by this invention judges whether the particulate alimentation of a particulate filter 30 exceeded the predetermined threshold first, as shown in drawing 7 (S102). For example, by judging whether the differential pressure (dP) of the entrance-side exhaust pressure of a particulate filter 30 and an outlet side exhaust pressure is larger than predetermined threshold differential pressure (dPr), particulate alimentation is measured indirectly. Next, it judges whether the exhaust gas temperature (Te) of a diesel power plant 20 is lower than predetermined postinjection initiation temperature (Ter) (S104). And more greatly than a threshold predetermined in particulate alimentation, when exhaust gas temperature is lower than postinjection initiation temperature, postinjection of a diesel power plant 20 is started (S106). Next, it judges whether the rotational frequencies of a diesel power plant 20 are below medium speed and the operational status of low loading (S108), and in corresponding, a measure required to prevent rotational frequency increase of a turbocharger is taken, and it opens fully the adjustable nozzle or way SUTOGETOBARUBU prepared in the turbocharger 10 (S110).

[0012]

[Embodiment of the Invention] Hereafter, the gestalt of operation of this invention is explained with reference to a drawing. In addition, in each drawing, the same or the explanation which attached the same sign or the similar sign at the corresponding member, and overlapped is omitted mutually. Drawing 1 is a block diagram explaining the gestalt of operation of the exhaust gas purge used for the diesel power plant of this invention.

[0013] In drawing, the fuel stored in the fuel tank 21 is stored in the fuel rail 23 with a pump 22. The fuel rail 23 is also called a common rail, it is a form independent of an engine speed and the fuel amount of supply, and generates an injection pressure and has heightened the average injection pressure. An injector 24 injects the opening time defined by ECU (Electric Control Unit) and the fuel oil consumption which becomes settled from the fuel pressure from the fuel rail 23 to the combustion chamber of a diesel power plant 20.

[0014] The turbocharger 10 is equipped with the exhaust air drive turbine 11 which transforms the energy in the exhaust gas of a diesel power plant 20 into mechanical energy, and the compressor 12 which compresses inhalation air. While specific fuel consumption is remarkably improved with the engine using a turbocharger 10 as compared with the natural aspiration engine of the same output, exhaust gas concentration is also improved. A shaft 13 conducts rotation of the exhaust air drive turbine 11 to a compressor 12.

[0015] An inlet manifold 15 is a duct for supplying the air compressed by the compressor 12 to the combustion chamber of a diesel power plant 20. An exhaust manifold 16 is a duct for supplying exhaust gas to the exhaust air drive turbine 11 from the combustion chamber of a diesel power plant 20. The intake throttle 17 is the throttle valve of the inhalation air of a compressor 12. EGR valve 18 is a bulb which is prepared in the last section of an EGR (Exhaust Gas Recirculation) path, and controls the amount of exhaust gas recycling. Here, EGR introduces and carries out recycling of a part of exhaust gas to an inhalation network, by mixing in gaseous mixture, lowers combustion temperature and reduces the yield of Nox. DPF30 is a particulate filter, needs to remove the deposited particulate at fixed spacing, and needs to reproduce a particulate filter while it carries out uptake of the particulate contained in exhaust gas.

[0016] ECU is equipped with the program required for the electronic circuitry as an exhaust gas purification control unit 40, or actuation of a microprocessor while it performs the combustion control of a diesel power plant 20. The exhaust gas purification control unit 40 equips the alimentation judging means 42, the exhaust gas temperature judging means 44, the postinjection control means 46, the operational status detection means 48, the exhaust gas temperature maintenance means 50, and the list with the EGR control section 52.

[0017] The alimentation judging means 42 judges whether the particulate alimentation of a particulate filter 30 exceeded the predetermined threshold, and judges whether the differential pressure dP of the entrance-side exhaust pressure of a particulate filter 30 and an outlet side exhaust pressure is larger than the predetermined threshold differential pressure dPr. The predetermined threshold differential pressure dPr judges that the particulate amount deposited on the particulate filter 30 reached to the reproductive complement. the differential pressure dP of the entrance-side exhaust pressure of a particulate filter 30, and an outlet side exhaust pressure -- differential pressure sensors, such as a semi-conductor type, -- dependability -- it can measure highly. In addition, the particulate alimentation of a particulate filter 30 may be replaced with a differential pressure type, and you may measure by the weight formula, and a deposition condition may be checked by looking using a visual sensor and it may measure that the particulate amount of the exhaust gas which penetrated the particulate filter 30 further is not decreasing enough.

[0018] It judges whether the exhaust gas temperature judging means 44 has exhaust gas temperature Te of a diesel power plant 20 lower than the predetermined postinjection initiation temperature Ter. Measurement of exhaust gas temperature Te is measured by the sensor from which resistance changes according to temperature like a resistance bulb, and is measured with the thermometer of other formats like an infrared type thermometer. The postinjection initiation temperature Ter is defined on the basis of exhaust gas temperature required for playback of a particulate filter 30.

[0019] The particulate alimentation of a particulate filter 30 exceeds a predetermined threshold, and the postinjection control means 46 starts postinjection of a diesel power plant 20, when the exhaust gas temperature of a diesel power plant 20 is lower than postinjection initiation temperature. Drawing 2 is the timing chart of the fuel injection injected by the combustion chamber of a diesel power plant from an injector. There are pilot injection, Main injection, and postinjection as timing of fuel injection. The timing of fuel injection is defined by ECU.

[0020] The operational status detection means 48 detects the operational status of a diesel power plant 20, and the engine speeds of a diesel power plant 20 are below medium speed and the operational status of low loading like [at the time of an idling or downward slope transit], for example, it has detected it in the condition that the need of the high load is carried out to the diesel power plant 20 like [at the time of acceleration]. Drawing 3 is the explanatory view of the operational status of a diesel power plant, Torque Nm is shown on an axis of ordinate, and engine-speed rpm is shown on the axis of abscissa. a operating range E1 -- the high engine-speed field of a diesel power plant, and a list -- an inside low engine-speed field -- it is -- the crown -- the usual operating range of the diesel power plant at the time of running a truck and a bus is shown including the torque field. The rotational frequencies of a diesel power

plant 20 are below medium speed and the operational status of low loading like an idling or moderation operation at the time of a light load in the medium-speed low torque field E2.

[0021] It returns to drawing 1 again and explanation is continued. The exhaust gas temperature raised by postinjection prevents it being changed into the mechanical energy of a turbocharger 10, and falling again, burns the particulate deposited on the particulate filter 30, and reproduces the exhaust gas temperature maintenance means 50. Since exhaust gas temperature is lower than the temperature suitable for playback of a particulate filter when an engine load like an idling is light, it is necessary to raise exhaust gas temperature to the temperature suitable for playback of a particulate filter. Then, it checks whether the rotational frequencies of a diesel power plant 20 are below medium speed and the operational status of low loading with the operational status detection means 48. When judged as the operational status by which the above output is needed to some extent for a diesel power plant 20 with the operational status detection means 48, it increases and the rotational frequency of a turbocharger 10 is made into high power.

[0022] When it is checked that the rotational frequencies of a diesel power plant 20 are below medium speed and the operational status of low loading and postinjection is further performed by the operational status detection means 48, the exhaust gas temperature maintenance means 50 prevents rotational frequency increase of a turbocharger 10, and prevents rotational frequency increase of a turbocharger 10. You specifically open fully the adjustable nozzle or way SUTOGETOBARUBU prepared in the turbocharger 10, and it is made for the energy of exhaust gas not to make it consume to the turbine side of a turbocharger 10.

[0023] Drawing 4 is the block diagram of an adjustable blade type turbine. As for the adjustable blade type turbine, the guide blade 64 as an adjustable nozzle which can be adjusted is formed near the air inhalation opening 66 of the turbine housing 61. The guide blade 64 which can be adjusted is controlled according to the exhaust gas flow rate and gas rate which flow to a turbine impeller, and a turbine rotational frequency is maintained in a supercharge scope. Rotation of an adjust ring 62 can adjust easily the include angle of the guide blade 64 which can be adjusted. Adjustment of an include angle is performed by the cone roll cam 63 or the adjusting lever 65 attached to each blade.

[0024] Drawing 5 is the explanatory view of the charge pressure adjustment by way SUTOGETOBARUBU. Way SUTOGETOBARUBU 73 was formed in the exhaust pipe 74 which connects an engine 71 and a turbocharger 72, and has changed the condition of performing an air bypass and discharging the exhaust gas of an engine 71 in the direct open air without a turbocharger 72, and the condition of supplying a turbocharger 72.

[0025] Drawing 6 is drawing which explains relation with the working state of an exhaust gas temperature maintenance means to the existence list of exhaust gas temperature and postinjection. ** The bar graph shown by "W/o post injection" shows the condition of not performing postinjection, and exhaust gas temperature is the lowest. ** Although the condition of performing postinjection is shown and exhaust gas temperature is higher than the condition of not performing postinjection, the bar graph shown by "W/post injection" does not have the enough rise of exhaust gas temperature, in order that a turbocharger may work. ** The bar graph shown by "W/post injection & full open VNT or WG" shows the condition of opening fully the adjustable nozzle or way SUTOGETOBARUBU (WG) prepared in the turbocharger 10 by the exhaust gas temperature maintenance means while performing postinjection, and exhaust gas temperature is the highest. It becomes the temperature which exhaust gas temperature burns the particulate deposited on the particulate filter 30, and can reproduce by using together postinjection and an exhaust gas temperature maintenance means.

[0026] Thus, actuation of the constituted equipment is explained below. Drawing 7 is a flow chart explaining actuation of the equipment of drawing 1. First, the exhaust gas purification control device 40 starts the regeneration program of a particulate filter prepared in ECU a fixed period (S100). It judges whether the differential pressure (dP) of the entrance-side exhaust pressure of a particulate filter 30 and an outlet side exhaust pressure is larger than predetermined threshold differential pressure (dPr) (S102). Here, the particulate alimentation of a particulate filter 30 judges indirectly whether the predetermined threshold was exceeded by measuring the differential pressure of the entrance-side exhaust pressure of a particulate filter 30, and an outlet side exhaust pressure.

[0027] Next, it judges whether the exhaust gas temperature (Te) of a diesel power plant 20 is lower than predetermined postinjection initiation temperature (Ter) (S104). And more greatly than a threshold predetermined in particulate alimentation, when exhaust gas temperature is lower than postinjection initiation temperature, postinjection of a diesel power plant 20 is started (S106). Next, it judges whether the rotational frequencies of a diesel power plant 20 are below medium speed and the operational status of low loading (S108). Specifically, it judges whether there are few engine speeds (Rpm) rather than the predetermined engine speed (Rpmr) as which the total fuel injection quantity (Q) by the injector 24 was similarly beforehand determined by experiment few from the predetermined fuel oil consumption (Qr) beforehand defined by experiment. And in corresponding to S108, it opens fully the adjustable nozzle or way SUTOGETOBARUBU prepared in the turbocharger 10 (S110). And it escapes from processing (S112).

[0028] [Effect of the Invention] As explained above, since the exhaust gas purge of this invention judges whether the predetermined threshold used as those criteria for which the particulate alimentation of a particulate filter needs playback with an alimentation judging means was exceeded, the stage when a particulate filter should be reproduced can judge it exactly. Since postinjection of a diesel power plant is started by the postinjection control means while exhaust gas temperature Te of a diesel power plant judges whether it is lower than the predetermined postinjection initiation temperature Ter with an exhaust gas temperature judging means, exhaust gas temperature can be held to the suitable temperature for particulate filter playback. Since it is controlling so that the exhaust gas temperature which prevented engine-speed increase of a turbocharger and rose by postinjection is not lost with a turbocharger when the engine speed of a diesel power plant performs postinjection by below medium speed and the operational status of low loading with an exhaust gas temperature maintenance means, a particulate filter is reproducible.

[0029] For example, the time of an idling has the rotational frequency of a diesel power plant as below medium speed and operational status of low loading. For example, at the time of an idling, to being about 20 [mg/m³], a particulate yield is about 200 [mg/m³], and there is an about 10 times as many aperture as this depending on the operation mode of a diesel-power-plant car at the time of the maximum output. although the exhaust gas temperature at the time of an idling is markedly boiled as compared with 100-200 degrees C and 550-750 degrees C at the time of the maximum output and is low temperature, it can be made to go up to the temperature to which exhaust gas temperature of the operational status of below medium speed and low loading is made as for playback of a particulate filter by the engine speed of a diesel power plant by operation of this invention. Then, without being used for the work from which the heat energy by postinjection raises the rotational frequency of a turbine in the time of the low idling condition of an exhaust-gas temperature, or downward slope transit, the engine equipped with the turbocharger can also raise an exhaust-gas temperature to the activity temperature region of a catalyst by postinjection, and can reproduce a particulate filter compulsorily.

[Translation done.]

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DESCRIPTION OF DRAWINGS

[Brief Description of the Drawings]

[Drawing 1] It is a block diagram explaining the gestalt of operation of the exhaust gas purge used for the diesel power plant of this invention.

[Drawing 2] It is the timing chart of the fuel injection injected by the combustion chamber of a diesel power plant from an injector.

[Drawing 3] It is the explanatory view of the operational status of a diesel power plant.

[Drawing 4] It is the block diagram of an adjustable blade type turbine.

[Drawing 5] It is the explanatory view of the charge pressure adjustment by way SUTOGETO BARUBU.

[Drawing 6] It is drawing which explains relation with the working state of an exhaust gas temperature maintenance means to the existence list of exhaust gas temperature and postinjection.

[Drawing 7] It is a flow chart explaining actuation of the equipment of drawing 1.

[Description of Notations]

10 Turbocharger

20 Diesel Power Plant

21 Fuel Tank (Fuel Tank)

22 Pump (Pump)

23 Fuel Rail (Common Rail)

24 Injector (Injector)

30 Particulate Filter (DPF)

40 Exhaust Gas Purification Control Unit (ECU)

42 Alimentation Judging Means

44 Exhaust Gas Temperature Judging Means

46 PostInjection Control Means

48 Operational Status Detection Means

50 Exhaust Gas Temperature Maintenance Means

[Translation done.]

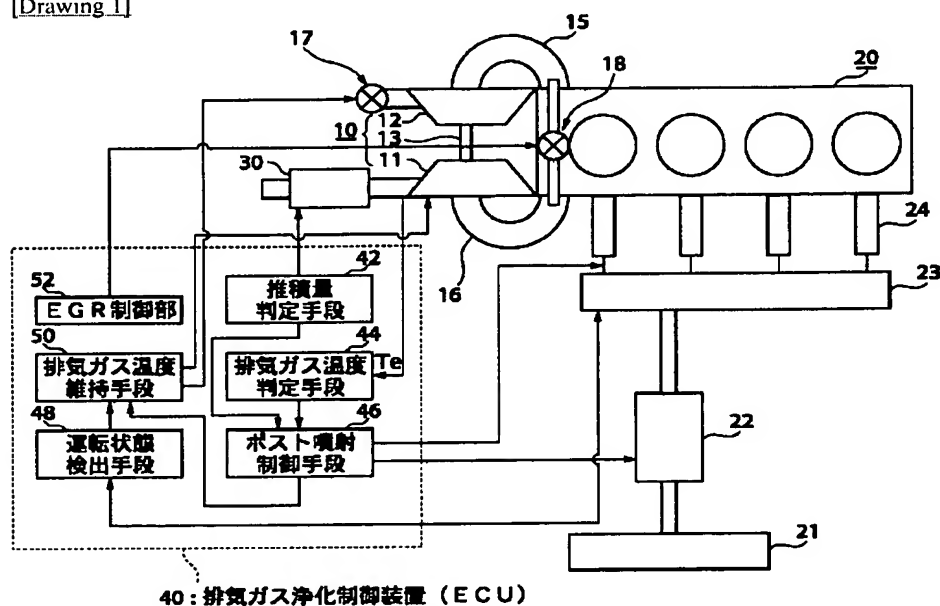
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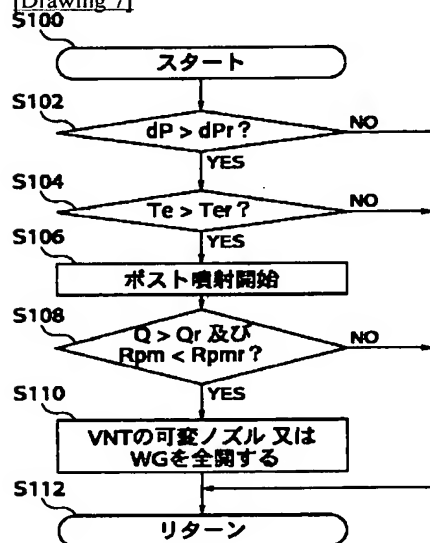
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DRAWINGS

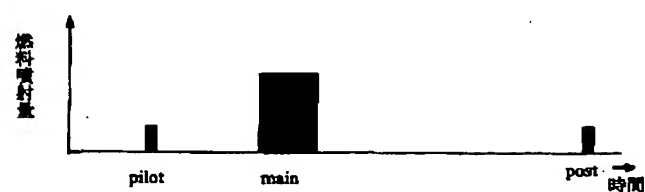
[Drawing 1]



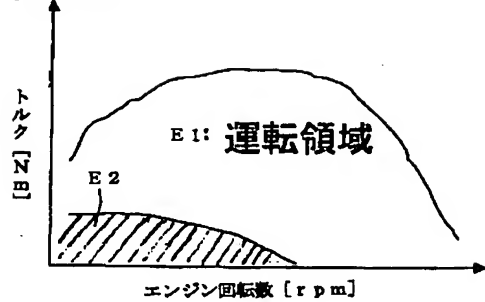
[Drawing 7]



[Drawing 2]



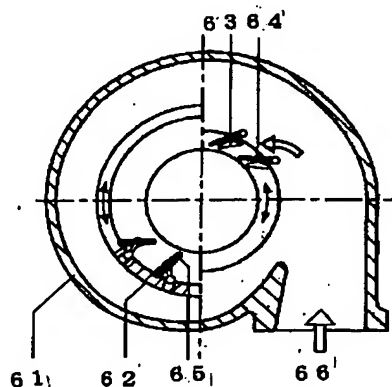
[Drawing 3]



[Drawing 4]

可変ブレード式タービンの構造

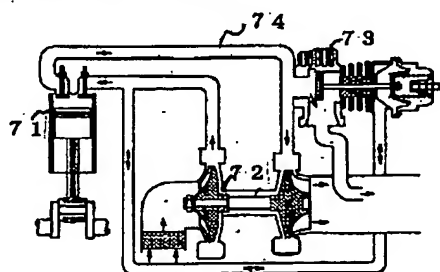
- 6 1: タービンハウジング
- 6 2: 調整リング
- 6 3: コントロールカム
- 6 4: 調整可能ガイドブレード
- 6 5: 調整レバー付きガイドブレード
- 6 6: エア吸入口



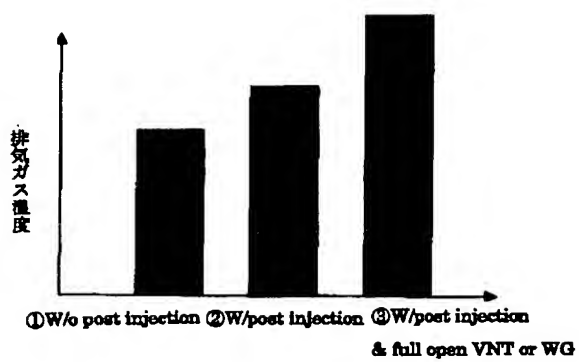
[Drawing 5]

ウェイストゲートバルブ による過給圧の調節

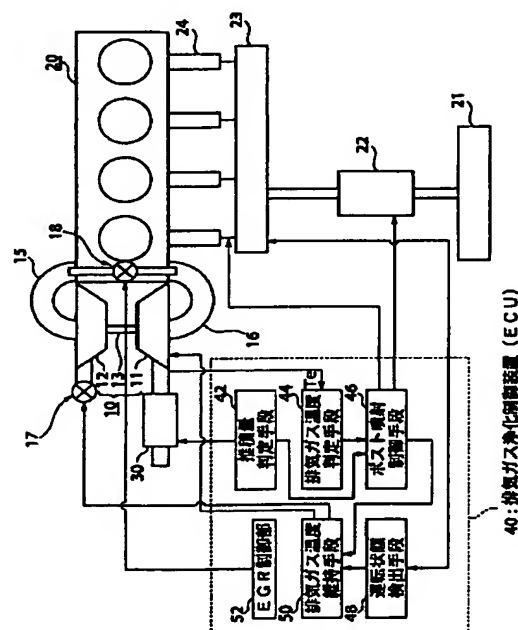
- 7 1: エンジン
- 7 2: ターボチャージャー
- 7 3: ウェイストゲートバルブ



[Drawing 6]



[Translation done.]



【特許請求の範囲】

【請求項1】 ターボチャージャーを有するディーゼルエンジンと、前記ディーゼルエンジンの排気ガスに含まれるバティキュレートを捕集するバティキュレート・フィルターを備え、前記バティキュレート・フィルターを前記排気ガスの熱エネルギーにより再生するエンジンに用いられる排気ガス浄化装置において、前記バティキュレート・フィルターのバティキュレート堆積量が所定の閾値を超えたか判断する堆積量判定手段と、

前記ディーゼルエンジンの排気ガス温度 (T_e) が所定のポスト噴射開始温度 (T_{er}) よりも低いか判断する排気ガス温度判定手段と；前記バティキュレート堆積量が所定の閾値を超え、かつ前記排気ガス温度が前記ポスト噴射開始温度より低い場合に、前記ディーゼルエンジンのポスト噴射を開始するポスト噴射制御手段と；前記ディーゼルエンジンの運転状態を検出する運転状態検出手段と；前記運転状態検出手段で検出した前記ディーゼルエンジンの回転数が中速以下且つ低負荷の運転状態でポスト噴射を実行する場合には、前記ターボチャージャーの回転数増大を阻止するように作動させる排気ガス温度維持手段とを具備する；排気ガス浄化装置。

【請求項2】 前記排気ガス温度維持手段は、前記ディーゼルエンジンの回転数が中速以下且つ低負荷の運転状態でポスト噴射を実行する場合には、前記ターボチャージャーに設けられた可変ノズル若しくはウェイトゲートバルブを全開する；請求項1に記載の排気ガス浄化装置。

【請求項3】 前記排気ガス温度維持手段を作動させる前記ディーゼルエンジンの回転数が中速以下且つ低負荷の運転状態が、前記ディーゼルエンジンのアイドルリングである；請求項1又は請求項2に記載の排気ガス浄化装置。

【請求項4】 前記堆積量判定手段が、バティキュレート・フィルターの入口側排気圧と出口側排気圧の差圧 (dP) が所定のしきい値差圧 (dP_r) よりも大きいか判断することからなる；請求項1乃至請求項3の何れかに記載の排気ガス浄化装置。

【請求項5】 ターボチャージャーを有するディーゼルエンジンと、前記ディーゼルエンジンの排気ガスに含まれるバティキュレートを捕集するバティキュレート・フィルターを備え、前記バティキュレート・フィルターを前記排気ガスの熱エネルギーにより再生するエンジンに用いられる排気ガス浄化方法において、前記バティキュレート・フィルターのバティキュレート堆積量が所定の閾値を超えたか判断し；前記ディーゼルエンジンの排気ガス温度 (T_e) が所定のポスト噴射開始温度 (T_{er}) よりも低いか判断し；前記バティキュレート堆積量が所定の閾値よりも大きく、かつ前記排気ガス温度が前記ポスト噴射開始温度よりは低い場合に、前記ディーゼルエンジンのポスト噴射を開始し；前記ディーゼルエン

ジンの回転数が中速以下且つ低負荷の運転状態か判断し；前記ディーゼルエンジンの回転数が中速以下且つ低負荷の場合には、前記ターボチャージャーの回転数増大を阻止する；排気ガス浄化方法。

【発明の詳細な説明】

【0001】

【発明の属する技術分野】本発明はディーゼルエンジンの排気ガスに含まれる微小な固体粒子（バティキュレート）を捕集して焼却する装置に関し、特にアイドルリングのような低負荷時にも効率良くDPF（ディーゼル・バティキュレート・フィルター）を再生する排気ガス浄化装置、排気ガス浄化方法に関する。

【0002】

【従来の技術】ディーゼルエンジンでは、噴射した燃料を圧縮着火させている。また不均質な混合気を燃焼すると必ず排気ガス中に微粒子が発生するため、原理的に煤煙のでないディーゼルエンジンを設計することは不可能に近い。しかし、ディーゼルエンジンでは、例えばターボチャージャー（特にインタークーラー付き）を採用することで、バティキュレートの排出を可視限界以下まで減少させている。

【0003】しかし、大都市における貨物輸送の需要増大は自動車の過密を招いている。そこで、一台一台のディーゼルエンジン機関が排気ガス対策を行っても、巨大な過密都市では多量のディーゼルエンジン機関が作動しているため、バティキュレートの総排出量が増大して環境負荷容量を超えてしまう場合が多い。そこで、ディーゼルエンジンの排気ガス中からバティキュレートを取り除くため、バティキュレート・フィルターがディーゼルエンジンを搭載した車両に装着されている。

【0004】バティキュレート・フィルターは、例えばハニカム構造のコーティングしていない多孔壁を備えている。排気ガスに含まれるバティキュレートは、多孔壁の孔部分に詰まることによって取り除かれる。捕集されたバティキュレートがバティキュレート・フィルターに堆積するので、この堆積したバティキュレートを一定間隔で除去してバティキュレート・フィルターを再生する必要がある。この再生方式として、例えば熱処理法が用いられている。熱処理法は高出力のバティキュレート燃焼用ヒータを接続して、排気ガスの温度を約700℃に上げて、バティキュレートを焼いてしまうものである。

【0005】しかし、この熱処理法はヒータの加熱エネルギーを必要とするため、燃費の悪化を招き、また再生と捕集の二系統からなるため装置が大きく複雑になるという課題がある。そこで、最近では連続再生型DPFと呼ばれる装置が盛んに開発されている。この装置は触媒によってバティキュレートの燃焼温度を下げ、排気ガスの熱エネルギーによって連続的に再生するものである。ヒータによる強制加熱ではなく、また二系統にする必要もないので、燃費・レイアウトの点で熱処理法を用いたD

PFより有利である。

【0006】

【発明が解決しようとする課題】ところが、上述の連続再生型DPFは触媒が活性化しない低温度域においては再生が行われない。従って、排気温度の低いアイドリング状態や下り坂走行時には再生されずに、パティキュレートがフィルターに堆積されつづけ、フィルターの目詰まりにより排圧が上昇し動力性能に支障をきたすという課題があった。そこで、このような場合にポスト噴射により排気温度を触媒の活性温度域まで上げて、強制的に再生することが行われる。しかるに、ターボチャージャーを備えたエンジンでは、せっかくポスト噴射によって排気温度を上げて、この熱エネルギーがタービンの回転数を上げる仕事に使われてしまい、タービンの出口温度すなわちDPF入口温度は低下してしまうという課題がある。本発明は上述する課題を解決するもので、ポスト噴射で発生した熱エネルギーを有効的に利用可能な排気ガス浄化装置を提供することを目的にしている。

【0007】

【課題を解決するための手段】上記目的を達成するために、本発明による排気ガス浄化装置は、例えば図1に示すように、ターボチャージャー10を有するディーゼルエンジン20と、ディーゼルエンジン20の排気ガスに含まれるパティキュレートを捕集するパティキュレート・フィルター30を備え、パティキュレート・フィルター30を前記排気ガスの熱エネルギーにより再生するエンジンに用いられる。

【0008】前記排気ガス浄化装置は、パティキュレート・フィルター30のパティキュレート堆積量が所定の閾値を超えたか判断する堆積量判定手段42と、ディーゼルエンジン20の排気ガス温度(Te)が所定のポスト噴射開始温度(Ter)よりも低いか判断する排気ガス温度判定手段44と、前記パティキュレート堆積量が所定の閾値を超え、かつ前記排気ガス温度が前記ポスト噴射開始温度より低い場合に、ディーゼルエンジン20のポスト噴射を開始するポスト噴射制御手段46と、ディーゼルエンジン20の運転状態を検出する運転状態検出手段48と、運転状態検出手段48で検出したディーゼルエンジン20の回転数が中速以下且つ低負荷の運転状態でポスト噴射を実行する場合には、ターボチャージャー10の回転数増大を阻止するように作動させる排気ガス温度維持手段50とを具備する。

【0009】このように構成された排気ガス浄化装置において、堆積量判定手段42は、パティキュレート・フィルター30のパティキュレート堆積量が所定の閾値を超えたか判断するもので、例えばパティキュレート・フィルター30の入口側排気圧と出口側排気圧の差圧(dP)が所定のしきい値差圧(dPr)よりも大きいと判断する構成とするといふ。排気ガス温度判定手段44は、ディーゼルエンジン20の排気ガス温度(Te)が

所定のポスト噴射開始温度(Ter)よりも低いか判断する。ポスト噴射制御手段46は、パティキュレート・フィルター30のパティキュレート堆積量が所定の閾値を超え、かつディーゼルエンジン20の排気ガス温度がポスト噴射開始温度より低い場合に、ディーゼルエンジン20のポスト噴射を開始して、排気ガス温度をパティキュレート・フィルター30の再生温度まで高くする。

【0010】運転状態検出手段48は、ディーゼルエンジン20の運転状態を検出するもので、例えばアイドリングのようにディーゼルエンジン20の回転数が中速以下且つ低負荷の運転状態であるか、加速時のようにディーゼルエンジン20に高い負荷が必要されている状態を検出している。排気ガス温度維持手段50は、ディーゼルエンジン20の回転数が中速以下且つ低負荷の運転状態で、ポスト噴射制御手段46によりポスト噴射が実行される場合には、ターボチャージャー10の回転数増大を阻止して、パティキュレート・フィルター30に堆積したパティキュレートを燃焼させて再生する。ターボチャージャー10の回転数増大を阻止する構成としては、例えばターボチャージャー10に設けられた可変ノズル若しくはウェイストゲートバルブを全開して、ターボチャージャー10のタービン側に排気ガスのエネルギーが消費させないようにする。他方、運転状態検出手段48でディーゼルエンジン20にある程度以上の出力が必要とされる運転状態と判断された場合は、ターボチャージャー10の回転数を増大して高出力としている。

【0011】上記目的を達成するために、本発明による排気ガス浄化方法は、例えば図7に示すように、まずパティキュレート・フィルター30のパティキュレート堆積量が所定の閾値を超えたか判断する(S102)。例えばパティキュレート・フィルター30の入口側排気圧と出口側排気圧の差圧(dP)が所定のしきい値差圧(dPr)よりも大きいと判断することで、パティキュレート堆積量を間接的に測定する。次に、ディーゼルエンジン20の排気ガス温度(Te)が所定のポスト噴射開始温度(Ter)よりも低いか判断する(S104)。そして、パティキュレート堆積量が所定の閾値よりも大きく、かつ排気ガス温度がポスト噴射開始温度よりは低い場合に、ディーゼルエンジン20のポスト噴射を開始する(S106)。次に、ディーゼルエンジン20の回転数が中速以下且つ低負荷の運転状態であるか判断し(S108)、該当する場合にはターボチャージャーの回転数増大を阻止するのに必要な措置をとるもので、例えばターボチャージャー10に設けられた可変ノズル若しくはウェイストゲートバルブを全開する(S110)。

【0012】

【発明の実施の形態】以下、本発明の実施の形態について、図面を参照して説明する。なお、各図において互いに同一あるいは相当する部材には同一符号または類似符

号を付し、重複した説明は省略する。図1は本発明のディーゼルエンジンに用いられる排気ガス浄化装置の実施の形態を説明する構成図である。

【0013】図において、燃料タンク21に蓄えられた燃料は、ポンプ22により燃料レール23に蓄えられる。燃料レール23はコモンレールとも呼ばれ、エンジン回転数及び燃料供給量に依存しない形で、噴射圧力を発生させて平均噴射圧力を高めている。インジェクタ24は、ECU(Electric Control Unit)により定められた開き時間と燃料レール23からの燃料圧力から定まる燃料噴射量を、ディーゼルエンジン20の燃焼室に噴射する。

【0014】ターボチャージャー10は、ディーゼルエンジン20の排気ガス中のエネルギーを機械エネルギーに変換する排気駆動タービン11と、吸入空気を圧縮するコンプレッサ12を備えている。同じ出力の自然吸気エンジンと比較して、ターボチャージャー10を用いたエンジンでは、燃料消費率が著しく改善されると共に、排気ガス濃度も改善される。シャフト13は、排気駆動タービン11の回転運動をコンプレッサ12に伝導する。

【0015】吸気マニホールド15は、コンプレッサ12で圧縮された空気をディーゼルエンジン20の燃焼室に供給する為の管路である。排気マニホールド16は、ディーゼルエンジン20の燃焼室から排気ガスを排気駆動タービン11に供給する為の管路である。インテーク・スロットル17は、コンプレッサ12の吸入空気の絞り弁である。EGRバルブ18は、EGR(Exhaust Gas Recirculation)通路の最終部に設けられ、排気ガス再循環量をコントロールするバルブである。ここで、EGRは排気ガスの一部を吸入系統に導入して再循環させ、混合気に混入することによって燃焼温度を下げ、 Nox の発生量を低減させるものである。DPF30はパティキュレート・フィルターで、排気ガスに含まれるパティキュレートを捕集すると共に、堆積したパティキュレートを一定間隔で除去してパティキュレート・フィルターを再生する必要がある。

【0016】ECUは、ディーゼルエンジン20の燃焼制御を行うと共に、排気ガス浄化制御装置40としての電子回路若しくはマイクロプロセッサの動作に必要なプログラムを備えている。排気ガス浄化制御装置40は、堆積量判定手段42、排気ガス温度判定手段44、ポスト噴射制御手段46、運転状態検出手段48、排気ガス温度維持手段50、並びにEGR制御部52を備えている。

【0017】堆積量判定手段42は、パティキュレート・フィルター30のパティキュレート堆積量が所定の閾値を超えたか判断するもので、例えばパティキュレート・フィルター30の入口側排気圧と出口側排気圧の差圧 dP が所定のしきい値差圧 dPr よりも大きいと判断す

る。所定のしきい値差圧 dPr は、パティキュレート・フィルター30に堆積したパティキュレートの量が、再生の必要な量まで達したことを判断するものである。パティキュレート・フィルター30の入口側排気圧と出口側排気圧の差圧 dP は、半導体式などの差圧センサによって信頼性高く測定できる。なお、パティキュレート・フィルター30のパティキュレート堆積量は差圧式に代えて、重量式で測定しても良く、また視覚センサを用いて堆積状態を視認しても良く、さらにパティキュレート・フィルター30を透過した排気ガスのパティキュレートの量が充分減少していないことを測定するものでも良い。

【0018】排気ガス温度判定手段44は、ディーゼルエンジン20の排気ガス温度 Te が所定のポスト噴射開始温度 Ter よりも低いと判断する。排気ガス温度 Te の測定は、例えば測温抵抗体のような温度に応じて抵抗値が変化するセンサで測定したり、また赤外線式温度計のような他の形式の温度計で測定する。ポスト噴射開始温度 Ter は、パティキュレート・フィルター30の再生に必要な排気ガス温度を基準に定める。

【0019】ポスト噴射制御手段46は、パティキュレート・フィルター30のパティキュレート堆積量が所定の閾値を超え、かつディーゼルエンジン20の排気ガス温度がポスト噴射開始温度より低い場合に、ディーゼルエンジン20のポスト噴射を開始する。図2はインジェクタからディーゼルエンジンの燃焼室に噴射される燃料噴射のタイミングチャートである。燃料噴射のタイミングとしては、パイロット噴射、メイン噴射、ポスト噴射がある。燃料噴射のタイミングはECUにより定められる。

【0020】運転状態検出手段48は、ディーゼルエンジン20の運転状態を検出するもので、例えばアイドリングや下り坂走行時のようにディーゼルエンジン20の回転数が中速以下且つ低負荷の運転状態であるか、加速時のようにディーゼルエンジン20に高い負荷が必要されている状態を検出している。図3はディーゼルエンジンの運転状態の説明図で、縦軸にトルク Nm 、横軸にエンジン回転数 rpm を示している。運転領域E1は、ディーゼルエンジンの高回転数領域、並びに中低回転数領域であって中高トルク領域を含むもので、トラックやバスが走行する際におけるディーゼルエンジンの通常の運転領域を示している。中速低トルク領域E2は、アイドリングや軽負荷時の減速運転のようにディーゼルエンジン20の回転数が中速以下且つ低負荷の運転状態である。

【0021】再び図1に戻って説明を継続する。排気ガス温度維持手段50は、ポスト噴射によって高められた排気ガス温度が、ターボチャージャー10の機械エネルギーに変換されて再び低下するのを防止して、パティキュレート・フィルター30に堆積したパティキュレート

を燃焼させて再生するものである。アイドリングのようなエンジン負荷の軽い場合には、排気ガス温度がパティキュレート・フィルターの再生に適する温度よりも低い。ため、排気ガス温度をパティキュレート・フィルターの再生に適する温度まで上昇させる必要がある。そこで、運転状態検出手段48によってディーゼルエンジン20の回転数が中速以下且つ低負荷の運転状態であるか否かを確認する。もし、運転状態検出手段48でディーゼルエンジン20にある程度以上の出力が必要とされる運転状態と判断された場合は、ターボチャージャー10の回転数を増大して高出力としている。

【0022】排気ガス温度維持手段50は、運転状態検出手段48によってディーゼルエンジン20の回転数が中速以下且つ低負荷の運転状態であると確認され、さらにポスト噴射が実行される場合には、ターボチャージャー10の回転数増大を阻止して、ターボチャージャー10の回転数増大を阻止する。具体的には、ターボチャージャー10に設けられた可変ノズル若しくはウェイトゲートバルブを全開して、ターボチャージャー10のタービン側に排気ガスのエネルギーが消費させないようにする。

【0023】図4は可変ブレード式タービンの構成図である。可変ブレード式タービンは、タービンハウジング61のエア吸入口66の近傍に可変ノズルとしての調整可能ガイドブレード64が設けてある。タービンインペラに流れる排気ガス流量やガス速度に合わせて調整可能ガイドブレード64が制御され、タービン回転数が過給有効範囲に維持される。調整リング62の回転により、調整可能ガイドブレード64の角度を容易に調整できる。角度の調整はコントロールカム63、または夫々のブレードに付属している調整レバー65により行われる。

【0024】図5はウェイトゲートバルブによる過給圧調整の説明図である。ウェイトゲートバルブ73は、エンジン71とターボチャージャー72とを接続する排気管74に設けられたもので、エアバイパスを行ってエンジン71の排気ガスをターボチャージャー72を介さず直接外気に排出する状態と、ターボチャージャー72に供給する状態とを切替えている。

【0025】図6は排気ガス温度とポスト噴射の有無並びに排気ガス温度維持手段の稼働状態との関係を説明する図である。①“W/o post injection”で示される棒グラフは、ポスト噴射を行わない状態を示しており、排気ガス温度が最も低くなっている。②“W/post injection”で示される棒グラフは、ポスト噴射を行う状態を示しており、排気ガス温度がポスト噴射を行わない状態よりも高くなっているが、ターボチャージャーが稼働するため排気ガス温度の上昇が十分でない。③“W/post injection & full open VNT or WG”で示される棒グラフは、ポスト噴射を行うと共に、排気ガス温度維持手段に

よりターボチャージャー10に設けられた可変ノズル若しくはウェイトゲートバルブ(WG)を全開する状態を示しており、排気ガス温度が最も高くなっている。ポスト噴射と排気ガス温度維持手段を併用することで、排気ガス温度がパティキュレート・フィルター30に堆積したパティキュレートを燃焼させて再生できる温度となる。

【0026】このように構成された装置の動作について、次に説明する。図7は図1の装置の動作を説明する流れ図である。まず、排気ガス浄化制御装置40はECUに設けられたパティキュレート・フィルターの再生処理プログラムを一定周期で起動する(S100)。パティキュレート・フィルター30の入口側排気圧と出口側排気圧の差圧(dP)が所定のしきい値差圧(dPr)よりも大きいか判断する(S102)。ここでは、パティキュレート・フィルター30の入口側排気圧と出口側排気圧の差圧を測定することで、間接的にパティキュレート・フィルター30のパティキュレート堆積量が所定の閾値を超えたか判断している。

【0027】次に、ディーゼルエンジン20の排気ガス温度(Te)が所定のポスト噴射開始温度(Ter)よりも低いか判断する(S104)。そして、パティキュレート堆積量が所定の閾値よりも大きく、かつ排気ガス温度がポスト噴射開始温度よりは低い場合に、ディーゼルエンジン20のポスト噴射を開始する(S106)。次に、ディーゼルエンジン20の回転数が中速以下且つ低負荷の運転状態であるか判断する(S108)。具体的には、予め実験によって定められた所定燃料噴射量(Qr)よりもインジェクタ24による全燃料噴射量(Q)が少なく、且つ、同じく実験によって予め定められた所定エンジン回転数(Rpmr)よりもエンジン回転数(Rpm)が少ないか判断している。そして、S108に該当する場合にはターボチャージャー10に設けられた可変ノズル若しくはウェイトゲートバルブを全開する(S110)。そして、処理を抜ける(S112)。

【0028】

【発明の効果】以上説明したように、本発明の排気ガス浄化装置は、堆積量判定手段によりパティキュレート・フィルターのパティキュレート堆積量が再生を必要とするかの基準となる所定の閾値を超えたか判断するので、パティキュレート・フィルターを再生すべき時期が的確に判断できる。排気ガス温度判定手段でディーゼルエンジンの排気ガス温度Teが所定のポスト噴射開始温度Terよりも低いか判断すると共に、ポスト噴射制御手段によりディーゼルエンジンのポスト噴射を開始するので、排気ガス温度をパティキュレート・フィルター再生に好適な温度に保持できる。排気ガス温度維持手段により、ディーゼルエンジンの回転数が中速以下且つ低負荷の運転状態でポスト噴射を実行する場合には、ターボ

ャージャーの回転数増大を阻止して、ポスト噴射により上昇した排気ガス温度がターボチャージャーで失われなように制御しているので、パティキュレート・フィルターの再生が行える。

【0029】ディーゼルエンジンの回転数が中速以下且つ低負荷の運転状態として、例えばアイドリング時がある。パティキュレートの発生量は、例えばアイドリング時に約20[mg/m³]であるのに対して、最大出力時に約200[mg/m³]であり、ディーゼルエンジン車両の運転モードに依存して約10倍の開きがある。アイドリング時の排気ガス温度は例えば100～200℃と、最大出力時の550～750℃と比較して格段に低温であるが、本発明の実施によりディーゼルエンジンの回転数が中速以下且つ低負荷の運転状態の排気ガス温度をパティキュレート・フィルターの再生ができる温度まで上昇させることができる。そこで、ターボチャージャーを備えたエンジンでも、排気温度の低いアイドリング状態や下り坂走行時において、ポスト噴射による熱エネルギーがタービンの回転数を上げる仕事に使われることなく、ポスト噴射により排気温度を触媒の活性温度域まで上げて、強制的にパティキュレート・フィルターを再生することができる。

【図面の簡単な説明】

【図1】 本発明のディーゼルエンジンに用いられる排気ガス浄化装置の実施の形態を説明する構成図である。*

*【図2】 インジェクタからディーゼルエンジンの燃焼室に噴射される燃料噴射のタイミングチャートである。

【図3】 ディーゼルエンジンの運転状態の説明図である。

【図4】 可変ブレード式タービンの構成図である。

【図5】 ウェイストゲートバルブによる過給圧調整の説明図である。

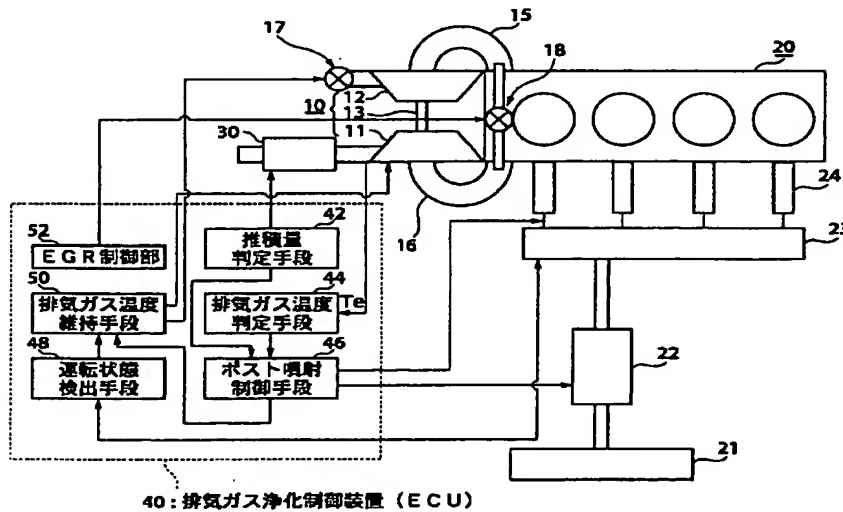
【図6】 排気ガス温度とポスト噴射の有無並びに排気ガス温度維持手段の稼働状態との関係を説明する図である。

【図7】 図1の装置の動作を説明する流れ図である。

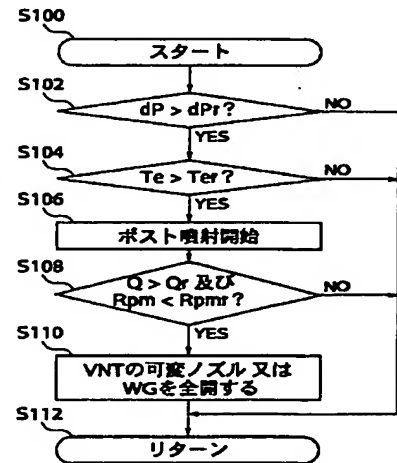
【符号の説明】

- 10 ターボチャージャー
- 20 ディーゼルエンジン
- 21 燃料タンク (Fuel Tank)
- 22 ポンプ (Pump)
- 23 燃料レール (Common Rail)
- 24 インジェクタ (Injector)
- 30 パティキュレート・フィルター (DPF)
- 40 排気ガス浄化制御装置 (ECU)
- 42 堆積量判定手段
- 44 排気ガス温度判定手段
- 46 ポスト噴射制御手段
- 48 運転状態検出手段
- 50 排気ガス温度維持手段

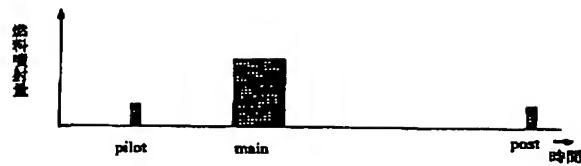
【図1】



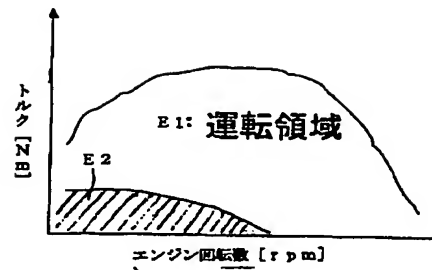
【図7】



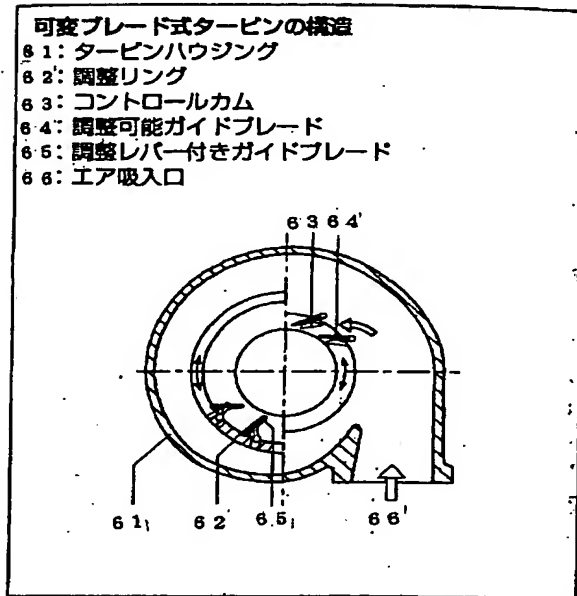
【図2】



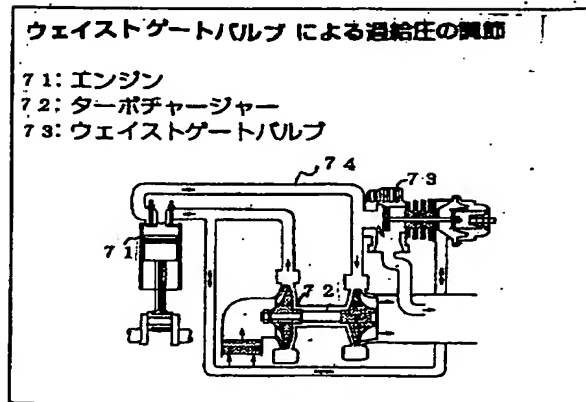
【図3】



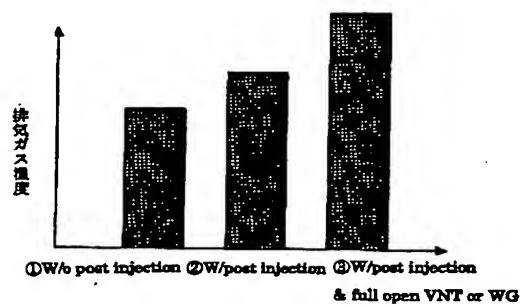
【図4】



【図5】



【図6】



フロントページの続き

(51)Int.Cl. ⁷	識別記号	F I	ターム (参考)
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			J
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		41/04	3 7 5
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 GA04 GB24 GB28 GD12 GD13
 GD17 GE09 HA05 HA12 HA18
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 FA06 HA05 JA03
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 BA15 BA20 BA24 CA03 CA09
 DA10 EB12 EC03 FA12 FA13
 FA17 FA27 FA33 FA37
 3G090 BA01 CA01 DA04 DA12 DA18
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